Engineering Note
for
E906 Detector Assembly

PROJECT: E906

TITLE: Station 2Y Hodoscope Assembly

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DATE: March 16, 2011

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ABSTRACT: This document describes an aluminum framework designed to secure and install a hodoscope array in E906. Once assembled this framework will be attached to an I-beam and hung in the E906 beamline.

DESIGN:

The hodoscope arrays for the station 2Y hodoscope are provided by UIUC. Each array is composed of 19 scintillators/phototubes mounted to a C-channel aluminum frame. The C-channels (McMaster Carr part number 9001K13) are composed of 6063 aluminum with a base width of 4”, leg length of 1”, and a wall thickness of 0.125”. The channels are secured to each other at each end with aluminum blocks and ¼-20 screws. Figure 1 shows the empty frame and Figure 2 shows the fully assembled hodoscope array.

For the Station 2Y hodoscope, two of these arrays will be placed with ends overlapping to form a detector approximately 95” x 84”. When placed into the beamline the scintillators will be oriented horizontally. The arrays will be held in place by a framework consisting of aluminum extrusions and fastening hardware. The extrusions and fasteners, produced by 8020 Inc., are shown in Figure 3 and the final assembly for the E906 beamline is shown in Figure 4. The perimeter of the framework is constructed from 1030 extrusions. The horizontal members are attached to the vertical members via the 4015 corner brackets (Fig. 4). The arrays will be secured to the framework by clamping the connector blocks on the C-channels, in 4 locations, to the horizontal members via the 4119 corner brackets (Fig. 4). The top of the 8020 framework is attached to an aluminum I-beam (S8 x 6.35) which will be used to hang the detector in the E906 beamline.

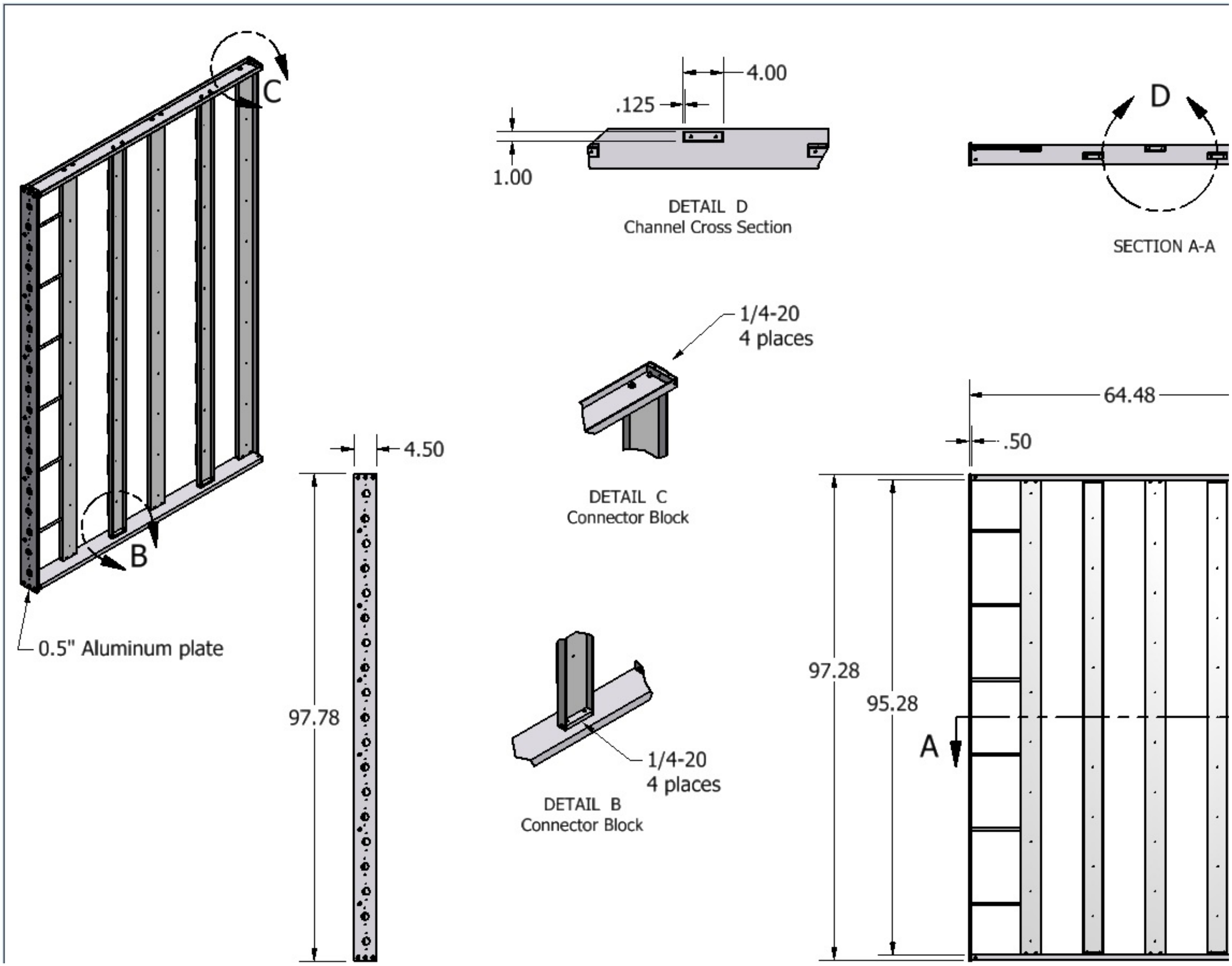
During assembly, the hodoscope arrays, 8020 extrusions, and I-beam will be laid out horizontally. After assembly the entire package shown in Figure 4 will be lifted and rotated to be installed vertically into the E906 beamline. Table 1 lists the mechanical properties of the extrusion profiles relevant to this report.

Part No.	Cross Section	Area (in^2)	Material	lbs/foot	Ix (in^4)	Iy (in^4)
1030	1" x 3"	1.1596	6105-T5*	1.3498	0.9711	0.1238

- UTS = 38ksi minimum, Yield = 35ksi minimum, Emod = 10.2e6 psi,
- Allowable Tensile Stress = 19.5ksi*

*(per Aluminum Design Manual Part IA, 1994).

Table 1 – Mechanical Properties of 8020 Extrusions in 1X Hodoscope Assembly per 8020 Fractional Parts Catalog – 16th edition


C
B
0.5" Aluminum plate
4.00
.125
1.00
DETAIL D
Channel Cross Section
D
SECTION A-A
1/4-20
4 places
DETAIL C
Connector Block
4.50
97.78
1/4-20
4 places
DETAIL B
Connector Block
64.48
.50
97.28
95.28
A

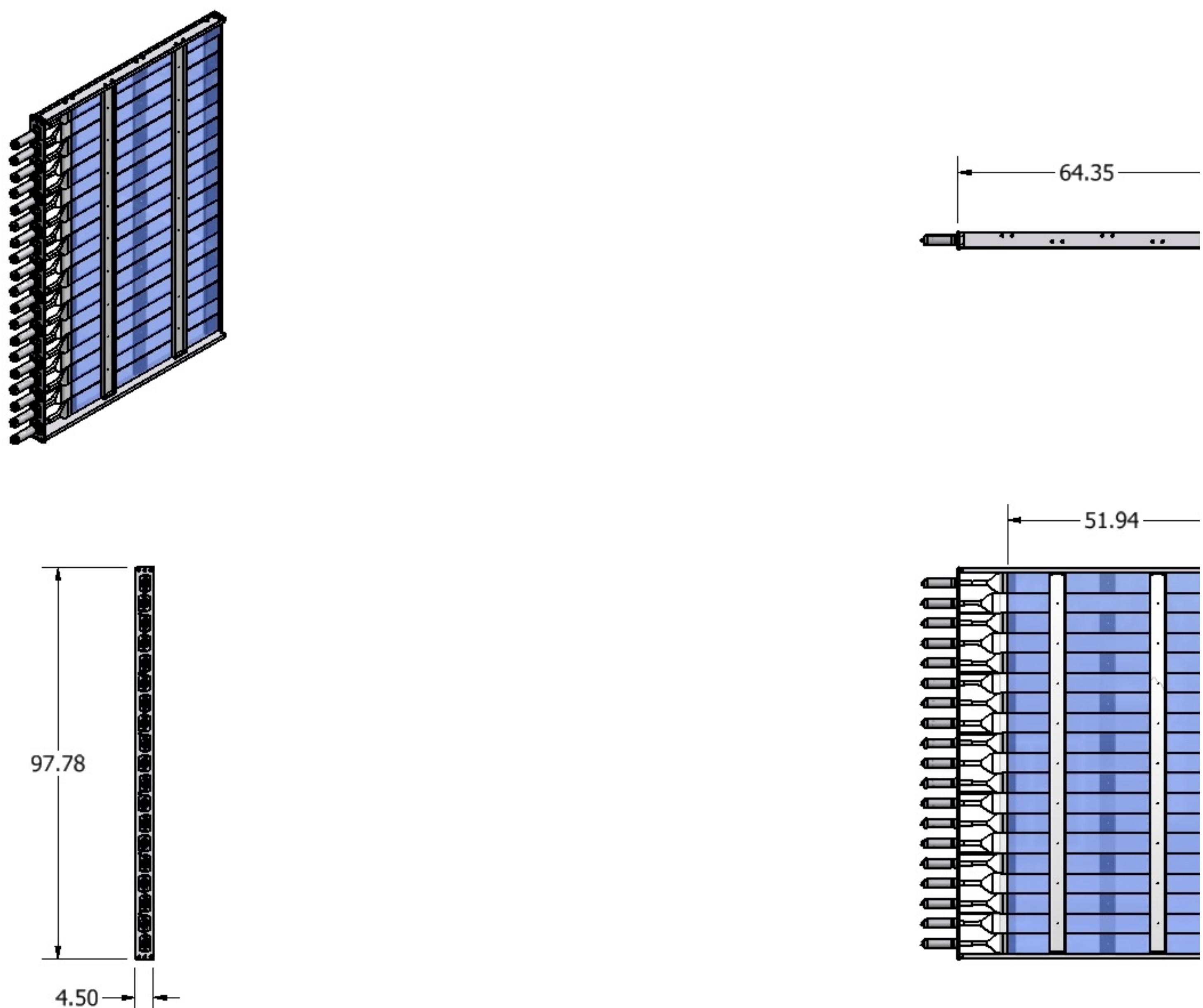


Figure 2: Station 2Y Hodoscope Array
(19 scintillators total)

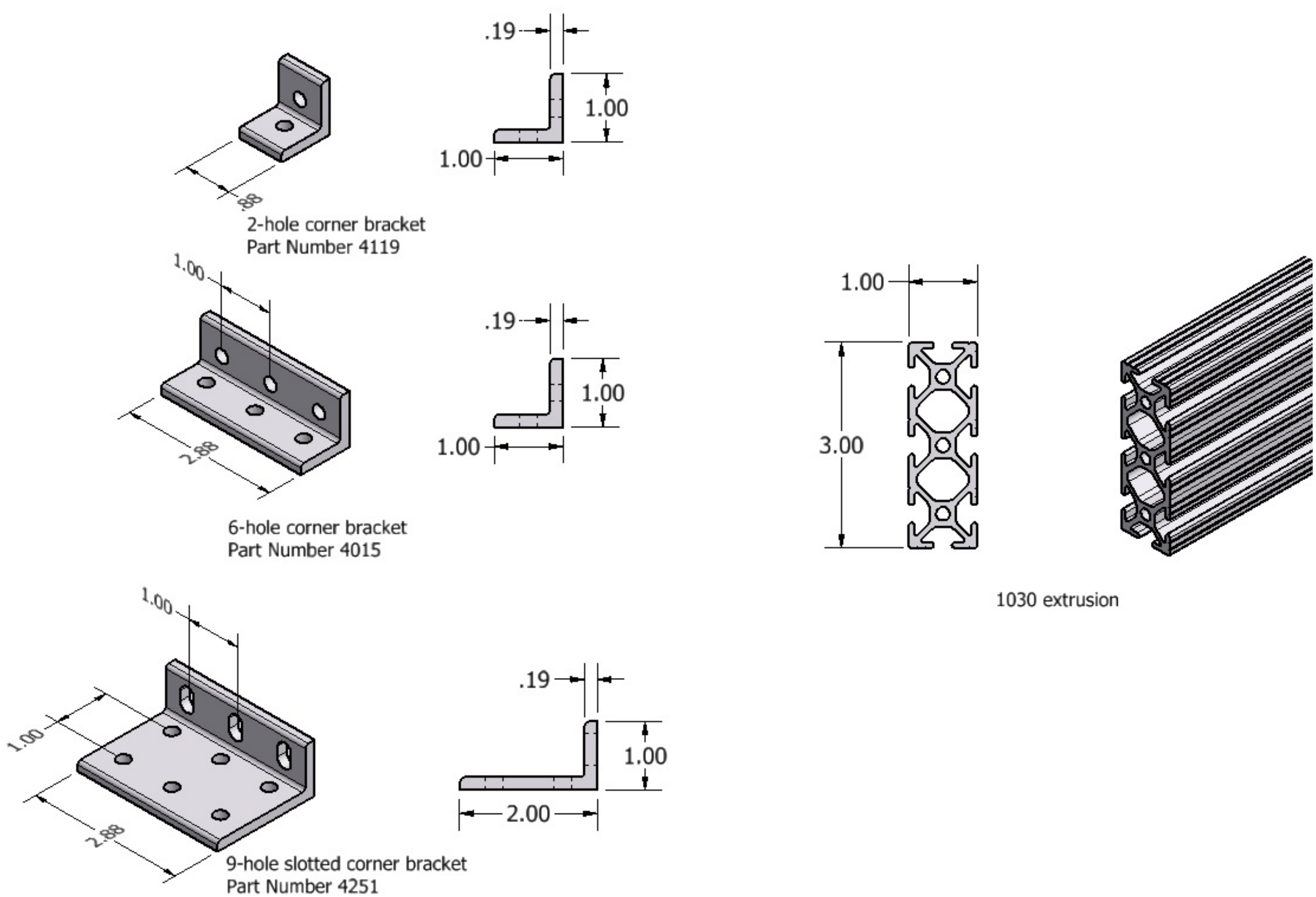


Figure 3: 8020 components used in 2Y Hodoscope Assembly

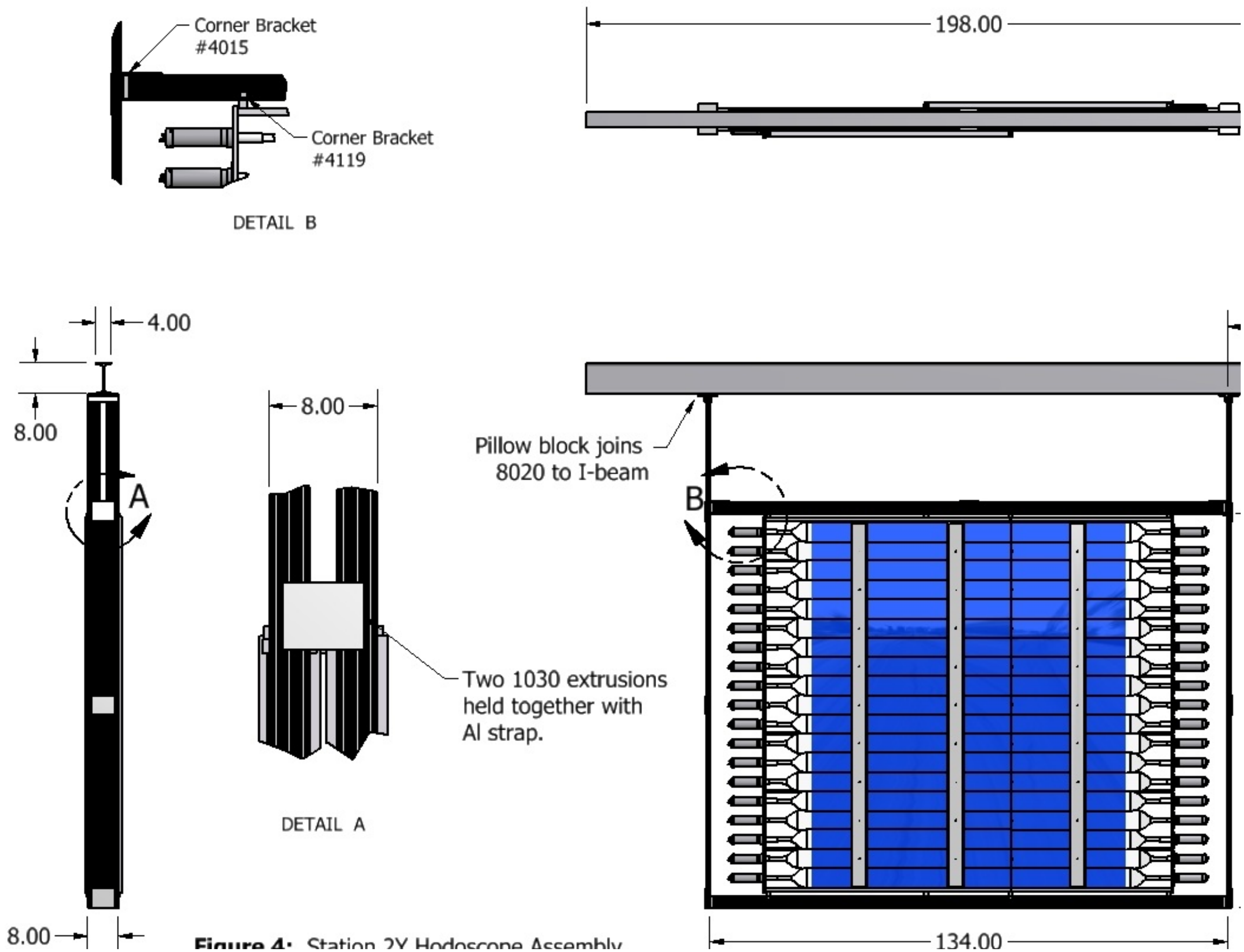


Figure 4: Station 2Y Hodoscope Assembly

ANALYSIS:
The hodoscope array will be built horizontally and then lifted/rotated and installed vertically in the E906 beamline with the tubes oriented horizontally. This will be done by wrapping slings around the I-beam and using the crane in NM4. The assembly must be strong enough to withstand the rotation from horizontal to vertical and also be strong enough to hang vertically for the duration of the experiment. Each of these cases is treated separately as follows:

Lifting/Rotating:
When the array assembly is horizontal it can be lifted at one edge and re-oriented to the vertical. If the assembly is supported along the top and bottom edges then the weight of the scintillators is essentially borne by two pairs of 1030 extrusions. The total thickness of the framework is 8 inches and the separation between the centers of these extrusions is 5 inches, with one extrusion located 2.5 inches above the mid plane and the other extrusion located 2 .5 inches below the mid plane. By substituting the appropriate values from Table 1 into the Parallel Axis Theorem, the resulting moment of inertia of each 1030 extrusion can be calculated as follows:

Parallel Axis Theorem: $I_z = I_x + A*r^2$ ***(1)***

Where: I_z is the moment of inertia about the parallel axis
I_x is the moment of inertia of the 1030 extrusion (Table 1)
A is the area of the 1030 extrusion (Table 1)
r is the distance to the parallel axis (2.5")

$I_z = 0.9711\text{in}^4 + (1.1596\text{in}^2)*(2.5\text{in})^2 = 8.2186\text{in}^4$

Since there are two extrusions along each edge the total value for I is twice the value calculated above, or 16.4372in^4.

If treated as beams supported on both ends subject to a concentrated load at the location shown in Figure 4, then the stress and deflection of each load can be calculated using standard formulas:

Stress at any point along the beam: $s = \frac{-Wb}{ZL}x;$ for $0 \le x \le a$ **(2)**

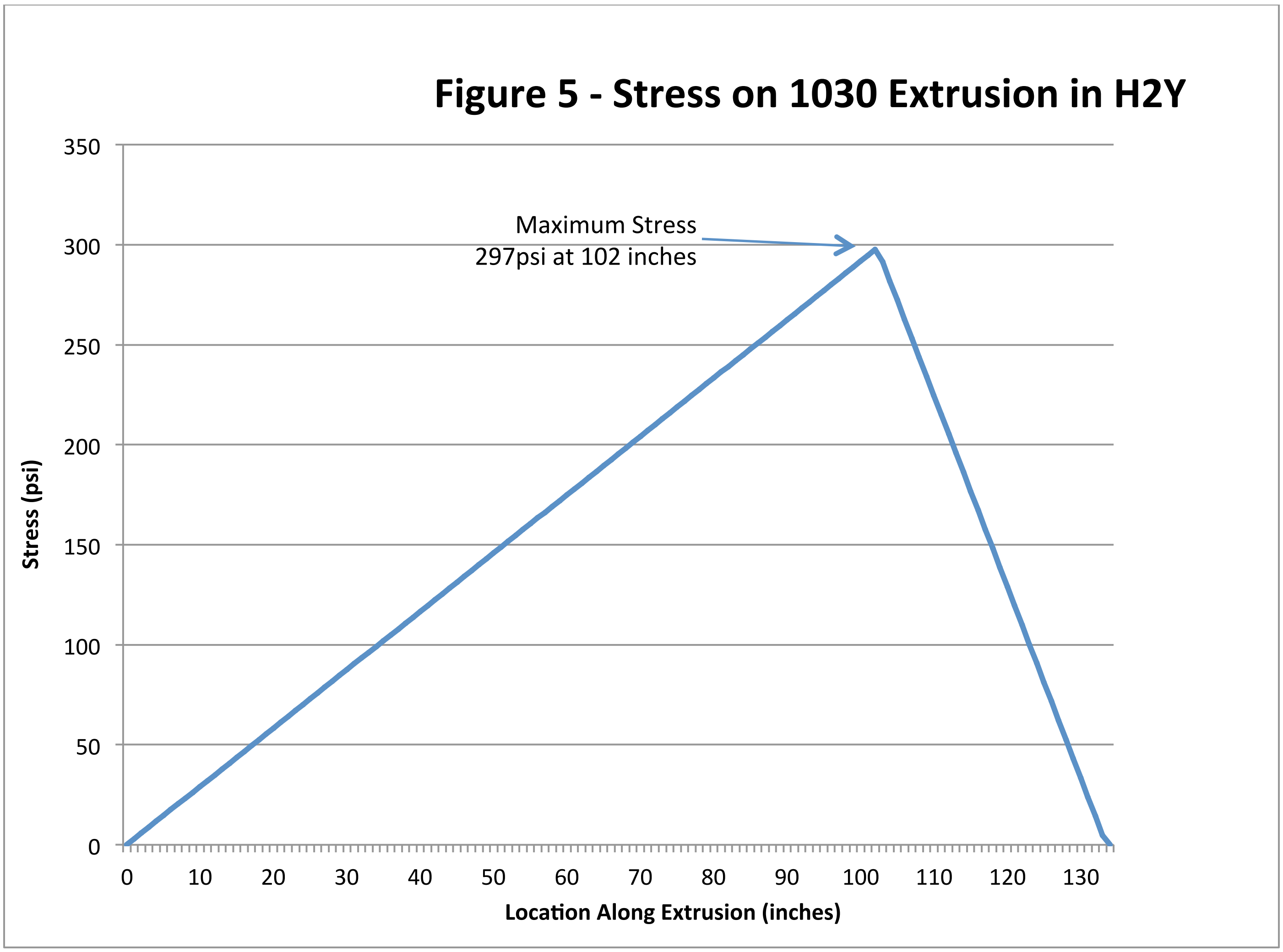
$s = \frac{-Wa}{ZL}(L-x);$ for $a \le x \le L$ **(3)**

Deflection at any point: $y = \frac{Wbx}{6EIL}(L^2 - b^2 - x^2);$ for $0 \le x \le a$ **(4)**

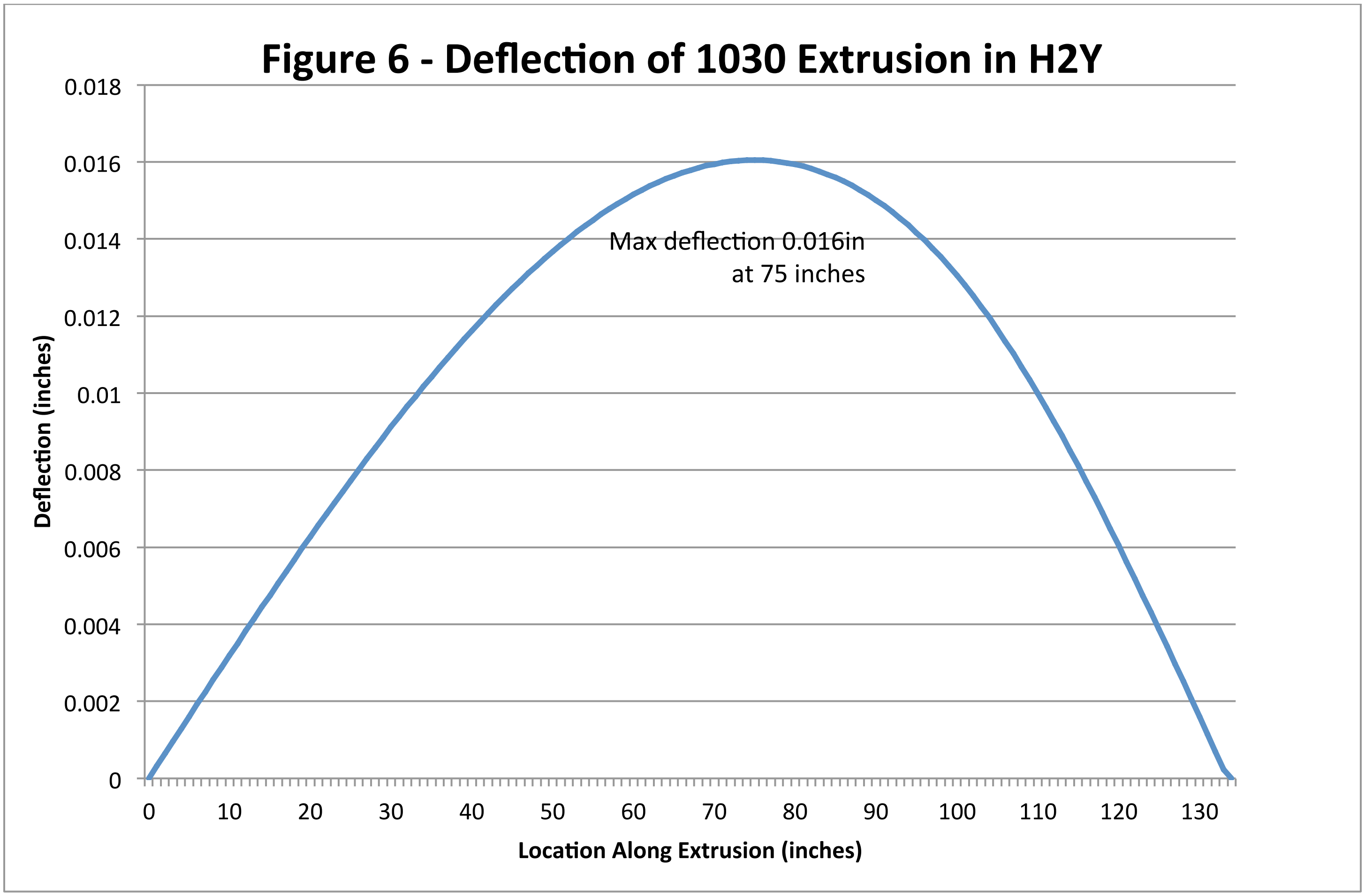
$y = \frac{Wa(L-x)}{6EIL}[L^2 - a^2 - (L-x)^2];$ for $a \le x \le L$ **(5)**

Where: *W* is the weight applied at the mounting point (82lb)
L is the length (133.5 inches)
a is the distance from the bottom end to the load
b is the distance from the load to the top end
x is the point in question
I is the combined moment of inertia, 16.4372in^4
Z is the section modulus (16.4372/2.5in = 3.2874in^3)
E is the modulus of elasticity

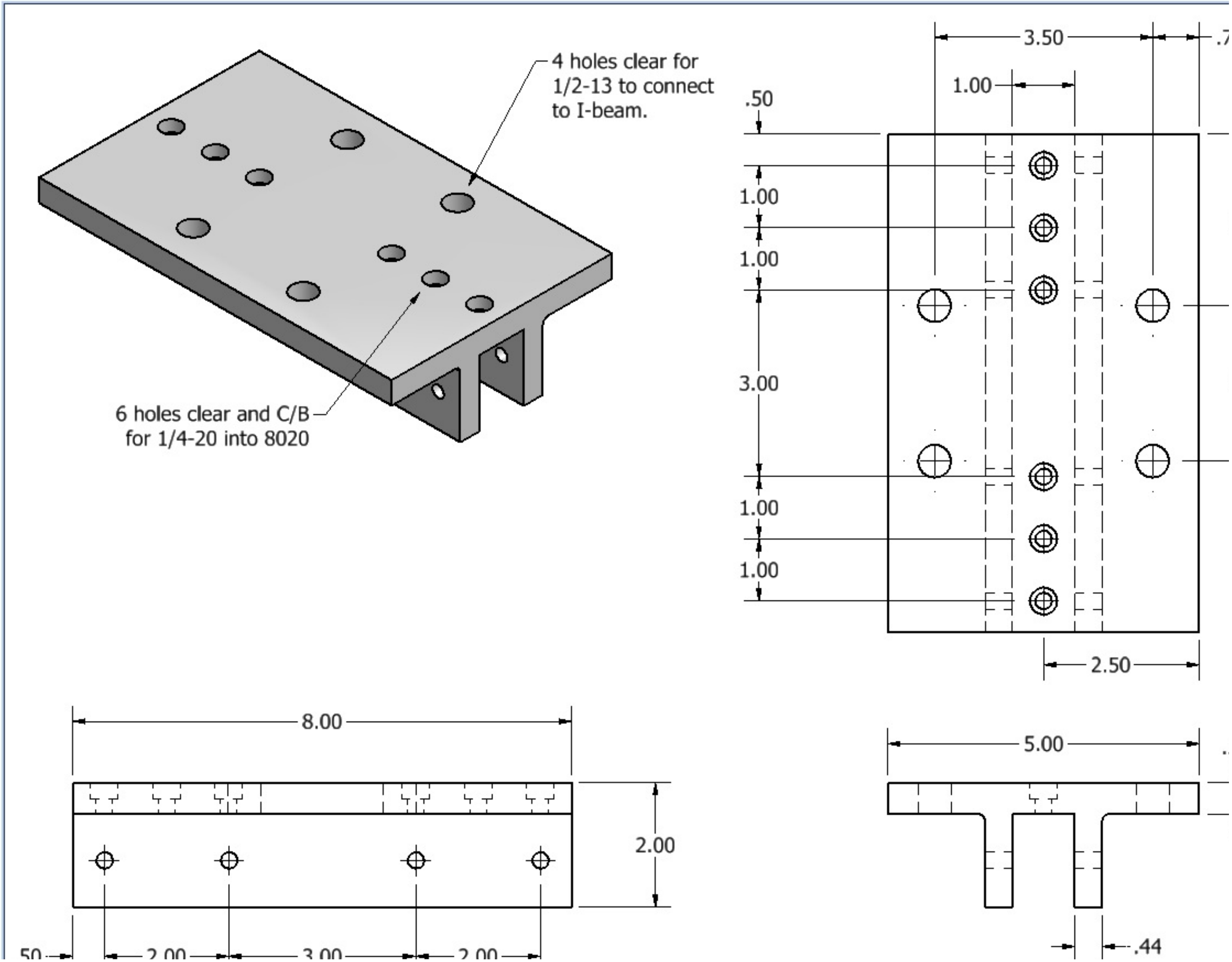
Solving Equations (2) and (3) for the load gives a maximum stress of 297 psi at a distance of roughly 102 inches from the bottom. A graph of this stresses is shown in Figure 5. The Allowable Bending Stress for a 1030 extrusion with a length of 133.5" (per Aluminum Design Manual Part IA, 1994) is 21,010psi. The maximum stress of 595psi, when compared to this value for Allowable Bending Stress, provides a safety factor of 21010/297 = 70.74 and is acceptable.



Likewise, solving Equations (4) and (5) for each individual load and adding those results gives a maximum deflection of 0.016 inches at a distance of 75 inches from the bottom. See Figure 6. This deflection occurs at the start of rotation and will fall to zero as the detector is made vertical and is acceptable.



Two aluminum pillow blocks are used to attach the vertical members of the hodoscope assembly to an aluminum I-beam (S8 x 6.35). See Figure 7. Each pillow block is attached to a pair of 1030 cross sections using six screws with ¼-20 threads. They are also attached to the I-beam using four 1/2-13 bolts. At the start of rotation each ¼-20 screw (12 screws total) will experience a shear of roughly 31.25-lbs which will vanish as the array is rotated. With a minor diameter of 0.1887 inches and an area of 0.0280-in^2, the resulting shear stress in each ¼-20 screw is 1116psi. Likewise, at the start of rotation each ½-13 bolt (8 bolts total) will experience a shear of roughly 46.875-lbs which will vanish as the array is rotated. With a minor diameter of 0.4041 inches and an area of 0.128-in^2, the resulting shear stress in each ½-13 screw is roughly 366psi. Grade 5 screws with yield strength of 92ksi (per SAE J429) are readily available. Assuming shear strength is 60% of yield strength results in shear strength of 55ksi which is far in excess of these expected actual values.


4 holes clear for
1/2-13 to connect
to I-beam.
6 holes clear and C/B
for 1/4-20 into 8020
3.50
1.00
.50
1.00
1.00
3.00
1.00
1.00
2.50
8.00
2.00
5.00
.44

Placement in E906 Beamline:
Once the detector assembly is vertical the weight of the hodoscopes is borne by four 1030 extrusion that run vertically along the edges of the framework. With an area of 1.1596in^2 (Table 1) and a weight per column of 93.75-lbs, the tensile stress on each vertical extrusion is 93.75/1.1596 = 80.8psi and is not a cause for concern when compared to the allowable tensile stress of 19.5ksi.

This hodoscope will be inserted into the beam line by resting the ends of the bottom surface of the aluminum I-beam onto the top surface of an A-Frame support structure. This A-Frame structure was built at Fermilab and analyzed in Engineering Note MSD-EN-1.1.3.8-KTeV. Once in place this aluminum I-beam will experience stress and deflection from the weight of the hodoscope. If treated as a beam supported on both ends subject to concentrated identical loads equidistant from center then the stress and deflection of the I-beam can be calculated using standard formulas:

Stress at center of constant cross section: $$s = \frac{-Wa}{Z} \quad \textbf{(5)}$$

Maximum deflection at center: $$y = \frac{Wa}{24EI}(3L^2 - 4a^2) \quad \textbf{(6)}$$

Where:
- W is the weight of each load (187.5lb)
- L is the length of the beam(198 inches)
- a is the distance from the end to the load (32 inches)
- I is the moment of inertia of S8x6.35 beam (57.6in^4)
- Z is the section modulus
- E is the modulus of elasticity

Substituting the values from Table 1 into equation (5) yields:

$$s = -\frac{187.5lb \times 32in}{\left(57.6in^4 \big/ 4in\right)} = -416.7lb/in^2$$

Likewise, substituting the values into equation (6) yields:

$$y = \frac{1}{24}\left[\frac{187.5lb \times (32in)}{10.2e6psi \times 57.6in^4}\right]\left[3(198in)^2 - 4(32in)^2\right] = 0.048in$$

The bending stress of 416.7psi and deflection of 0.048-in of the S8x6.35 I-beam are not a cause for concern.

Finally, in the beam line the weight of the entire array/framework assembly is held by the twelve ¼-20 screws and eight 3/8-16 in tension. Each ¼-20 screw experiences a tensile force of roughly 31.25-lbs. With a tensile stress area of 0.0318in^2, the resulting tensile stress in each ¼-20 screw is 983psi. Likewise, each ½-13 screw experiences a tensile force of roughly 46.875-lbs. With a tensile stress area of 0.1419in^2, the resulting tensile stress in each ½-13 screw is 330psi. These are acceptable stresses for Grade 5 screws per SAE J429.